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Soviet-era science, translated into English

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1957

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**Abstract**

**Full Text**

**THEORY OF ELASTICITY**

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**APPROXIMATE SOLUTION OF ELASTO-PLASTIC PROBLEMS OF THE THEORY OF IDEAL PLASTICITY**

*(Presented by Academician A. I. Nekrasov, 12 X 1956)*

We shall seek the solution of the elastoplastic problem in the form of series in powers of a certain parameter  $\delta$ :

$$\sigma_{ij} = \sum_{n=0} \delta^n \sigma_{ij}^{(n)}, \quad \sigma_{ii} \equiv \sigma_\rho, \quad \sigma_{jj} \equiv \sigma_\theta, \quad \sigma_{ij} \equiv \sigma_{ji} \equiv \tau_{\rho\theta}. \quad (1)$$

We shall confine ourselves to consideration, in the theory of plane strain, of the plasticity conditions of Mises and Saint-Venant <sup>(1)</sup>, which are essentially coincident:

$$\frac{1}{4}(\sigma_\rho - \sigma_\theta)^2 + \tau_{\rho\theta}^2 = 1; \quad (2)$$

in the theory of plane stress—the Saint-Venant plasticity condition <sup>(1)</sup>:

$$\frac{1}{4}(\sigma_\rho - \sigma_\theta)^2 + \tau_{\rho\theta}^2 = \left[1 - \frac{1}{2}(\sigma_\rho + \sigma_\theta)\right]^2 \quad \text{for } \sigma_\theta > \sigma_\rho > 0. \quad (3)$$

In (1), (2), (3) all quantities are dimensionless. Substituting (1) into (2), with  $\tau_{\rho\theta}^0 = 0$ , we obtain:

$$\sigma'_\rho - \sigma'_\theta = 0, \quad (\sigma''_\rho - \sigma''_\theta)\mu + \tau_{\rho\theta}'^2 = 0, \quad \frac{1}{2}(\sigma'''_\rho - \sigma'''_\theta)\mu + \tau'_{\rho\theta}\tau''_{\rho\theta} = 0, \quad (4)$$

where  $\mu = \text{sign}(\sigma_\rho^0 - \sigma_\theta^0)$ .

Substituting (1) into (3) with  $\tau_{\rho\theta}^0 = 0$ , we obtain:

$$\sigma'_\theta = 0, \quad (1 - \sigma_\rho^0)\sigma''_\theta + \tau_{\rho\theta}'^2 = 0, \quad (1 - \sigma_\rho^0)\sigma'''_\theta - \sigma'_\rho\sigma''_\theta + 2\tau'_{\rho\theta}\tau''_{\rho\theta} = 0. \quad (5)$$

If on the boundary  $L$  of the body under consideration

$$\sigma_n = P_n, \quad \tau_n = P_\tau, \quad (6)$$

then, assuming the equation of the contour  $L$  to be given in the form

$$\rho = \sum_{n=0}^{\infty} \delta^n \rho_n(\theta),$$

we obtain the linearized boundary conditions (6) in the form

$$\begin{aligned} \sigma'_\rho + \frac{d\sigma_\rho^0}{d\rho} \rho_1 &= \frac{dP_n}{d\rho} \rho_1; & \tau'_{\rho\theta} - (\sigma_\theta^0 - \sigma_\rho^0) \dot{R}_1 &= \frac{dP_\tau}{d\rho} \rho_1; \\ \sigma''_\rho + \frac{d\sigma'_\rho}{d\rho} \rho_1 + \frac{d^2\sigma_\rho^0}{d\rho^2} \frac{\rho_1^2}{2!} + \frac{d\sigma_\rho^0}{d\rho} \rho_2 + (\sigma_\theta^0 - \sigma_\rho^0) \dot{R}_1^2 - 2\tau'_{\rho\theta} \dot{R}_1 &= \\ &= \frac{d^2P_n}{d\rho^2} \frac{\rho_1^2}{2!} + \frac{dP_n}{d\rho} \rho_2; \end{aligned}$$

$$\begin{aligned} \tau''_{\rho\theta} - (\sigma_\theta^0 - \sigma_\rho^0) (\dot{R}_2 - R_1 \dot{R}_1) - (\sigma'_\theta - \sigma'_\rho) \dot{R}_1 + \frac{d}{d\rho} [\tau'_{\rho\theta} - (\sigma_\theta^0 - \sigma_\rho^0) \dot{R}_1] \rho_1 &= \\ &= \frac{d^2P_\tau}{d\rho^2} \frac{\rho_1^2}{2!} + \frac{dP_\tau}{d\rho} \rho_2; \end{aligned} \quad (7)$$

$$\begin{aligned} \sigma''''_\rho + \frac{d\sigma''_\rho}{d\rho} \rho_1 + \frac{d^2\sigma'_\rho}{d\rho^2} \frac{\rho_1^2}{2!} + \frac{d^3\sigma_\rho^0}{d\rho^3} \frac{\rho_1^3}{3!} + 2(\sigma_\theta^0 - \sigma_\rho^0) (\dot{R}_1 \dot{R}_2 - R_1 \dot{R}_1^2) + \\ + \frac{d(\sigma_\theta - \sigma_\rho)}{d\rho} \dot{R}_1^2 \rho_1 - (\sigma'_\theta - \sigma'_\rho) \dot{R}_1^2 - 2\tau'_{\rho\theta} (\dot{R}_2 - R_1 \dot{R}_1) - \\ - 2 \frac{d\tau_{\rho\theta}}{d\rho} \dot{R}_1 \rho_1 - 2\tau''_{\rho\theta} \dot{R}_1 + \frac{d\sigma'_\rho}{d\rho} \rho_2 + \frac{d^2\sigma_\rho^0}{d\rho^2} \rho_1 \rho_2 + \frac{d\sigma_\rho^0}{d\rho} \rho_3 = \\ = \frac{d^3P_n}{d\rho^3} \frac{\rho_1^3}{3!} + \frac{d^2P_n}{d\rho^2} \rho_1 \rho_2 + \frac{dP_n}{d\rho} \rho_3; \end{aligned}$$

$$\begin{aligned}
 & \tau_{\rho\theta}''' - 2\tau_{\rho\theta}' \dot{R}_1^2 - (\sigma_\theta^0 - \sigma_\rho^0)(\dot{R}_3 - R_1 \dot{R}_2 + R_1^2 \dot{R}_1 - \dot{R}_1 R_2 - \dot{R}_1^3) - \\
 & - (\sigma_\theta' - \sigma_\rho')(\dot{R}_2 - R_1 \dot{R}_1) - (\sigma_\theta'' - \sigma_\rho'') \dot{R}_1 + \frac{d}{d\rho} [\tau_{\rho\theta}'' - (\sigma_\theta^0 - \sigma_\rho^0)(\dot{R}_2 - R_1 \dot{R}_1) - \\
 & - (\sigma_\theta' - \sigma_\rho') \dot{R}_1] \rho_1 + \frac{d^2}{d\rho^2} [\tau_{\rho\theta}' - (\sigma_\theta^0 - \sigma_\rho^0) \dot{R}_1] \frac{\rho_1^2}{2!} + \frac{d}{d\rho} [\tau_{\rho\theta}' - (\sigma_\theta^0 - \sigma_\rho^0) \dot{R}_1] \rho_2 = \\
 & = \frac{d^3 P_\tau}{d\rho^3} \frac{\rho_1^3}{3!} + \frac{d^2 P_\tau}{d\rho^2} \rho_1 \rho_2 + \frac{d P_\tau}{d\rho} \rho_3,
 \end{aligned}$$

where  $R_i = \rho_i / \rho_0$ .

Since on the boundary of the plastic region  $L_s$  the solutions for the plastic and elastic regions are matched continuously (1):

$$[\sigma_\rho] = [\sigma_\theta] = [\tau_{\rho\theta}] = 0 \quad \text{on } L_s, \quad (8)$$

then, representing the equation of the contour  $L_s$  in the form

$$\rho_s = \sum_{n=0} \delta^n \rho_{sn}(\theta), \quad (9)$$

we obtain the linearized matching conditions (8) in the form

$$\begin{aligned}
 & \left[ \sigma'_{ij} + \frac{d\sigma_{ij}^0}{d\rho} \rho_{s1} \right] = 0, \quad \left[ \sigma''_{ij} + \frac{d\sigma'_{ij}}{d\rho} \rho_{s1} + \frac{d^2 \sigma_{ij}^0}{d\rho^2} \frac{\rho_{s1}^2}{2!} + \frac{d\sigma_{ij}^0}{d\rho} \rho_{s2} \right] = 0; \\
 & \left[ \sigma'''_{ij} + \frac{d\sigma''_{ij}}{d\rho} \rho_{s1} + \frac{d^2 \sigma'_{ij}}{d\rho^2} \frac{\rho_{s1}^2}{2!} + \frac{d^3 \sigma_{ij}^0}{d\rho^3} \frac{\rho_{s1}^3}{3!} + \frac{d\sigma_{ij}^0}{d\rho} \rho_{s3} + \frac{d\sigma'_{ij}}{d\rho} \rho_{s2} + \right. \\
 & \quad \left. + \frac{d^2 \sigma_{ij}^0}{d\rho^2} \rho_{s1} \rho_{s2} \right] = 0 \text{ etc.}
 \end{aligned} \quad (10)$$

We note that from the equilibrium equations and conditions (8) it follows that

$$\left[ \frac{d\sigma_\rho^0}{d\rho} \right] = 0 \quad \text{on } L_s.$$

Consequently, the matching conditions (10) for  $\sigma_\rho$  and  $\tau_{\rho\theta}$  play the role of boundary conditions for determining the stresses in the elastic region, while the matching condition for  $\sigma_\theta$  serves to determine  $\rho_{sn}$ . The greatest difficulty and interest in such problems is the determination of the equation of the boundary of the plastic zone  $L_s$ . We shall present several approximate solutions.

1. **Biaxial tension of a thick plate with a circular hole of radius  $a$  by forces  $P_1$  and  $P_2$ .** Denoting

$$\delta = \frac{|P_1 - P_2|}{2k}, \quad (11)$$

where  $k = \frac{1}{4}\sigma_s$  according to Saint-Venant and  $k = \frac{1}{\sqrt{3}}\sigma_s$  according to Mises, one may obtain:

$$\begin{aligned} \rho_s = 1 + \delta \cos 2\theta - \frac{3}{4}\delta^2(1 - \cos 4\theta) + \frac{5}{8}\delta^3(-\cos 2\theta + \cos 6\theta) + \\ + \frac{7}{64}\delta^4(-1 - 4\cos 4\theta + 5\cos 8\theta) + \dots \end{aligned} \quad (12)$$

Expansion (12) coincides exactly with the expansion of the equation of an ellipse with semiaxes  $(1 + \delta)$ ,  $(1 - \delta)$ , which, as shown in (2), is the exact solution of this problem.

2. **Biaxial tension of a thin plate with a circular hole of radius  $a$  by forces  $P_1$  and  $P_2$ .** Introducing  $\delta$  (11), one may obtain:

$$\begin{aligned} \rho_s = 1 + 4\delta^* \cos 2\theta - 8\delta^{*2}(1 - 2\cos 4\theta) - 80\delta^{*3}(\cos 2\theta - \cos 6\theta) + \\ + 32\delta^{*4}(1 - 16\cos 4\theta + 14\cos 8\theta) + \dots, \end{aligned} \quad (13)$$

where  $\delta^* = \delta/\alpha$ ;  $\alpha = a/r_s^0$ ;  $r_s^0$  is the size of the radius of the plastic zone for  $\delta = 0$ . The first approximation (13) was obtained in (3).

3. **Biaxial tension of a thin plate with an elliptical hole by forces  $P_1 d_2$  and  $P_2 d_2$ , directed at an angle  $\theta_0$  to the principal central axes of the ellipse.** Introducing  $\delta$  (11), one may obtain:

$$\begin{aligned} \rho_s = 1 + \delta^*(4d_2 \cos 2(\theta - \theta_0) + 3\alpha d_1 \cos 2\theta) + \delta^{*2}\{d_1^2(\alpha^2/4 - 8\alpha^4) - \\ - (18d_1 d_2 \alpha \cos 2\theta_0 + 8d_2^2) + [-d_1^2(15/4\alpha^2 - 8\alpha^3 - 3/4\alpha^4) + \\ + (18d_1 d_2 \alpha \cos 2\theta_0 + 16d_2^2 \cos 4\theta_0)] \cos 4\theta + \\ + [18d_1 d_2 \alpha \sin 2\theta_0 + 16d_2^2 \sin 4\theta_0] \sin 4\theta\} + \dots, \end{aligned}$$

where the equation of the ellipse of the hole is represented in the form

$$\rho = \alpha + \delta\alpha d_1 \cos 2\theta - \delta^2 \frac{3\alpha d_1^2}{4}(1 - \cos 4\theta) + \dots$$

For  $d_1 = 0$ ,  $\theta_0 = 0$ ,  $d_2 = 1$  there is the case of biaxial tension of a thin plate with a circular hole; for  $d_2 = 0$ ,  $d_1 = 1$ , the case of uniform tension of a thin plate with an elliptical hole.

4. **An eccentric tube under the action of internal pressure  $p_0$ .** Referring all linear quantities to the outer radius of the tube  $b$ , denote  $\delta = c/b$ , where  $c$  is the eccentricity of the tube. One may obtain:

$$\rho_s = \beta_0 - \delta \frac{2\beta_0^4}{1 - \beta_0^4} \cos \theta + \delta^2 \left\{ \frac{\beta_0^3(2 - \beta_0^4 - \beta_0^6)}{(1 - \beta_0^2)(1 - \beta_0^4)^2} + \frac{2\beta_0^7}{(1 - \beta_0^4)^2} + \frac{1}{N} \left[ -\frac{(1 - \beta_0^2)(1 - 3\beta_0^4)}{\beta_0} + \frac{(1 - \beta_0^2)^2(5 + 3\beta_0^4)\beta_0}{(1 - \beta_0^4)} - \frac{\beta_0^3}{(1 - \beta_0^4)^2} [(1 + \beta_0^2)^4 + 4\beta_0^4(2 - \beta_0^2)^2 - 4(1 + 4\beta_0^4)] + \frac{2\beta_0^3(1 - \beta_0^2)^2}{(1 + \beta_0^2)^2} \right] \cos 2\theta \right\},$$

where  $\beta_0 = r_s^0/b$ ,  $N = (\beta_0 - 1/\beta_0)^4$ .

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Received  
9 X 1956

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*Note: Figure translations are in progress. See original paper for figures.*

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